55. Estimation of engine piston system wear using time-frequency method

Marek Flekiewicz¹, Paweł Fabiś², Rafał Burdzik³
Silesian University of Technology, Department of Automotive Vehicle Construction, Gliwice, Poland
¹Corresponding author
E-mail: ¹marek.flekiewicz@polsl.pl, ²pawel.fabis@polsl.pl, ³rafal.burdzik@polsl.pl
(Received 20 August 2014; received in revised form 8 September 2014; accepted 11 September 2014)

Abstract. The paper presents the fault diagnosis technique for internal combustion engines using time-frequency representations of vibration signal. Engine block vibration results as a sum of many excitations mainly connected with engine speed and their intensity increases with the appearance of a fault or in case of higher engine elements wearing. In this paper an application of acceleration signals for the estimation of the influence of piston skirt clearance on diesel engine block vibrations has been described. Engine body accelerations registered for three simulated cases representing piston skirt clearance variations were an object of preliminary analysis. The presented procedures were applied to vibration and pressure signals acquired for a 0.5 dm³ Ruggerini, air cooled diesel engine. Reciprocating machines are difficult to diagnose using traditional frequency domain techniques because of generate transient vibration. In conducted experiments wavelet transform has been chosen.

Keywords: piston slap, engine diagnostic, vibroacoustic diagnosis.

1. Introduction

Piston slap is one of the most characteristic sources of engine body vibrations. Intensity of that excitation and its variations for different engine cycles depends mainly on in-cylinder pressure alterations. Changes of piston slap force value, influencing the piston horizontal movements result also from following factors:
- piston and piston pin mass, connecting rod mass,
- dimensions and the geometry of crank-connecting rod mechanism,
- engine crankshaft angle,
- engine speed and its load,
- piston skirt clearance.

Engine block vibrations result as a sum of many excitations mainly connected with engine speed, and their intensity increases with the appearance of a fault or in case of higher engine elements wearing. There are several fault recognition methods currently in use, they are based on spectrum density analysis both in time and frequency domains, FFT, as well as on wavelet transform and Wigner-Ville transform [1-10]. For the engine block it is important the type of construction and material due to vibroactivity [11, 13, 14].

In this paper, acceleration signals were used to estimate the influence of piston skirt clearance on engine block vibrations. Values of piston slap force, responsible for piston movement, were estimated on the basis of dynamic models of piston-connecting rod mechanism and the in-cylinder pressure variations in the function of engine crank angle. Engine body vibration signal, registered for three simulated cases representing piston skirt clearance variations, was an object of preliminary analysis.

2. Experimental setup

The procedures presented in this paper were applied to vibration and pressure signals from a 0.5 dm³ Ruggerini, air cooled diesel engine.

Test program provided acquisition of the following data:
- in-cylinder pressure,
- vibration signal of engine head and wall, for two directions: $x$ and $y$,
- crankshaft revolution, together with TDC recognition,
- engine torque,
- manifold pressure.

In-cylinder pressure was measured with the use of piezoelectric pressure transducer type 6121 by KISTLER, coupled with charge amplifier model 5011. Crankshaft position and TDC recognition was measured with the use of KISTLER 2613B transducer. Engine body vibrations were measured with ICP sensors by PCB interfacing with PA3000 signal conditioner manufactured by Roga Instruments.

![Fig. 1. Schematic diagram of experimental setup](image)

3. Analysis of results

The analysis was completed in a way allowing the identification of two characteristic ranges of vibration signal around TDC [12]:
- first range – from 300° to 372° crank angle,
- second from 372° to 420° crank angle.

First range includes information about initiation and running of the combustion process. In the second range is appeared temporary, notable increased of acceleration vibration amplitude, which describes the piston slap phenomena.

Crank angle where the temporary increased of the acceleration vibration amplitude occurs caused by piston slap phenomena is a function of clearance.

Take in the consideration that described crank angle range contain phenomena of the vibration amplitude appear which were piston slap response. For three received clearance values moment when the amplitudes appear include in 372-420° CA range. It has received that the separate crank angle range identifies mechanical phenomenas during the working cycle of engine.

Because vibration acceleration amplitudes bind with piston slap phenomena had very high values. Those calculations were carried out independent for first and second crank angle range.

Figures present crank angle-scale representation for three different simulated clearance values. Trace obtained for nominal clearance are presented on Fig. 2, meanwhile Fig. 3 presents results for 2 times bigger clearance and figure 4 for the 4 times bigger clearance. Changes in signal traces due to bigger clearance may also be noticed in time frequency plane.
Accelartion signals registrated for three different clearances value have been analyzed with the use of CWT (Continues Wavelet Transform). First, searched the right level of scale which could separate components of mechanical phenomena from components of combustion process and enable to identify of simulated piston clearance value.

Fig. 2. CWT for nominal clearance

Fig. 3. CWT for two Times bigger nominal clearance

Fig. 4. CWT for four times bigger nominal clearance
For signals after wavelet transformation were carried out calculation of wavelet coefficient derivative $C'_{ab}$. Results shown on the Figs. 5 and 6.

Derivative of wavelet coefficient distributions are shown that the maximal value of frequency for different piston skirt clearance value tend to move in the direction of higher frequency values. It also shown that the vibration signals have higher $C'_{ab}$ value together with increase of piston skirt clearance.

4. Conclusions

The wavelet transform is powerful tool for on-line monitoring and diagnostic of combustion process and engine systems estimation. It can recover important features of the vibration signal that are sensitive to the change of IC engine condition. That is the reason to apply the CWT transform.

Based on the results authors have proposed detection and piston skirt clearance monitoring algorithm. Scheme of that algorithm is shown on the Fig. 7.
55. ESTIMATION OF ENGINE PISTON SYSTEM WEAR USING TIME-FREQUENCY METHOD.
MAREK FLEKIEWICZ, PAWEŁ FABIŚ, RAFAL BURDZIK

Fig. 7. Engine piston system condition algorithm

References


